

## Transient Vibration

Impulse force

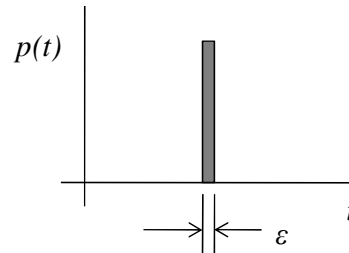
$$\hat{F} = \int p(t) dt$$

Delta function

$$\delta(t - \tau) = \begin{cases} \text{non zero} & t = \tau \\ 0 & t \neq \tau \end{cases}$$

$$\int_0^{\infty} \delta(t - \tau) dt = 1$$

$$\text{and } \int_0^{\infty} p(t) \delta(t - \tau) dt = p(\tau)$$



Solution of undamped system:

$$x(t) = \frac{\dot{x}(0)}{\omega_n} \sin \omega_n t + x(0) \cos \omega_n t$$

Since an impulse force  $\hat{F}$  acting at small time may present an initial velocity  $\frac{\hat{F}}{m}$ , then

$$x(t) = \frac{\hat{F}}{m\omega_n} \sin \omega_n t = \hat{F}h(t)$$

Where  $h(t)$  is the unit impulse response

$$h(t) = \frac{1}{m\omega_n} \sin \omega_n t$$

For damped system: 
$$x(t) = \frac{\hat{F} e^{-\zeta \omega_n t}}{m\omega_n \sqrt{1 - \zeta^2}} \sin \omega_n \sqrt{1 - \zeta^2} t = \hat{F}h(t)$$

**For Arbitrary Excitation:**

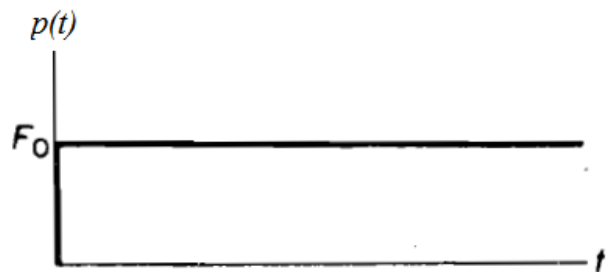
Given the response for unit impulse force is known  $h(t)$ , the response for an arbitrary force  $p(t)$  can be estimated by using superposition integral or convolution integral (Duhamel Integral):

$$x(t) = \int_0^t p(\tau)h(t - \tau)d\tau = \frac{1}{m\omega_n} \int_0^t p(\tau) \sin \omega_n(t - \tau)d\tau$$

The above integral state that the overall response is the sum of responses to forces each force is acting at different time  $\tau$  and have duration  $d\tau$ .

**Example:**

Determine the response of undamped system to the force shown in the figure:



Soultion:

$$h(t) = \frac{1}{m\omega_n} \sin \omega_n t$$

$$\begin{aligned} x(t) &= \int_0^t p(t)h(t - \tau)d\tau = \int_0^t \frac{F_0}{m\omega_n} \sin \omega_n(t - \tau)d\tau \\ &= \frac{F_0}{k} (1 - \cos \omega_n t) \end{aligned}$$

**Laplace transform Method**

And so, we can evaluate the response due to ramp or parabolic function. Laplace transform can also be used to find the transient response.

**Numerical Methods**

When the integral is not possible in closed form, then numerical methods such as Euler and Runge-Kutta methods can be used.

### Numerical Integration

The response to an arbitrary excitation  $p(t)$  with zero initial condition is given by

$$\begin{aligned} x(t) &= \int_0^t p(\tau)h(t-\tau)d\tau = \frac{1}{m\omega_n} \int_0^t p(\tau) \sin \omega_n(t-\tau)d\tau \\ &= \sin \omega_n t \frac{1}{m\omega_n} \int_0^t p(\tau) \cos \omega_n \tau d\tau - \cos \omega_n t \frac{1}{m\omega_n} \int_0^t p(\tau) \sin \omega_n \tau d\tau \\ &= A(t) \sin \omega_n t - B(t) \cos \omega_n t \end{aligned}$$

Where:

$$\begin{aligned} A(t) &= \frac{1}{m\omega_n} \int_0^t p(\tau) \cos \omega_n \tau d\tau \\ B(t) &= \frac{1}{m\omega_n} \int_0^t p(\tau) \sin \omega_n \tau d\tau \end{aligned}$$

Can be evaluated numerically. Consider  $A(t)$ :

$$A(t) = \frac{1}{m\omega_n} \int_0^t p(\tau) \cos \omega_n \tau d\tau = \frac{1}{m\omega_n} \int_0^t y(\tau) d\tau = \frac{\Delta\tau}{m\omega_n} \frac{1}{\mu} T_\mu^t$$

Where

$$\begin{aligned} T_1^t &= y_0 + y_1 + y_2 + \dots + y_{N-1} && \text{Simple Integration} \\ T_2^t &= y_0 + 2y_1 + 2y_2 + \dots + 2y_{N-1} + y_N && \text{Trapizoidal Integration} \\ T_3^t &= y_0 + 4y_1 + 2y_2 + \dots + 4y_{N-1} + y_N && \text{Simpson Integration} \end{aligned}$$

Using Incremental integration for numerical solution:

$$T_1^t = T_1^{t-\Delta\tau} + p(t-\Delta\tau) \cos \omega_n(t-\Delta\tau) \quad \text{Simple Integration}$$

$$T_2^t = T_2^{t-\Delta\tau} + p(t-\Delta\tau) \cos \omega_n(t-\Delta\tau) + p(t) \cos \omega_n t \quad \text{Trapizoidal}$$

$$T_3^t = T_3^{t-\Delta\tau} + [p(t-2\Delta\tau) \cos \omega_n(t-2\Delta\tau) + 4p(t-\Delta\tau) \cos \omega_n(t-\Delta\tau) + p(t) \cos \omega_n t] \quad \text{Simpson}$$

For damped vibration:

$$h(t) = \frac{e^{-\zeta\omega_n t}}{m\omega_d} \sin \omega_d t$$

$$x(t) = A(t)e^{-\zeta\omega_n t} \sin \omega_n t - B(t)e^{-\zeta\omega_n t} \cos \omega_n t$$

$$A(t) = \frac{1}{m\omega_n} \int_0^t p(\tau) e^{\zeta\omega_n \tau} \cos \omega_n \tau d\tau$$

$$B(t) = \frac{1}{m\omega_n} \int_0^t p(\tau) e^{\zeta\omega_n \tau} \sin \omega_n \tau d\tau$$

H.W: